

The following formulas cover the basic calculations used in brake application engineering.

Required	Given	Formula
Full load motor torque ( $T_{flmt}$ ), lb-ft	Horsepower (P), hp Shaft speed (N), rpm 5252 = Constant	$T_{flmt} = \frac{5252 \times P}{N}$
Average dynamic braking torque ( $T_d$ ), lb-ft	Total inertia reflected to brake ( $Wk^2$ ), lb-ft <sup>2</sup> Shaft speed at brake (N), rpm Desired stopping time (t), seconds 308 = Constant	$T_d = \frac{Wk^2 \times N}{308 \times t}$
Static torque (T), lb-ft	Force (F), lb Pulley or drum radius, (R), ft	$T = F \times R$
Overhauling dynamic torque reflected to brake shaft ( $T_o$ ), lb-ft	Weight of overhauling load (W), lb Linear velocity of descending load (V), ft/min Shaft speed at brake (N), rpm 0.158 = Constant	$T_o = \frac{0.158 \times W \times V}{N}$
Static torque of brake ( $T_s$ ), lb-ft (General Guideline)	Dynamic braking torque required ( $T_d$ ), lb-ft 0.8 = Constant (derating factor)	$T_s = \frac{T_d}{0.8}$
Inertia of rotating load reflected to brake shaft ( $Wk_b^2$ ), lb-ft <sup>2</sup>	Inertia of rotating load ( $Wk_L^2$ ), lb-ft <sup>2</sup> Shaft speed at load ( $N_L$ ), rpm Shaft speed at brake ( $N_B$ ), rpm	Equivalent $Wk_b^2 = Wk_L^2 \left( \frac{N_L}{N_B} \right)^2$
Equivalent inertia of linear moving load reflected to brake shaft ( $Wk_w^2$ ), lb-ft <sup>2</sup>	Weight of linear moving load (W), lb Linear velocity of load (V), ft/min Shaft speed at brake ( $N_B$ ), rpm $2\pi$ = Constant	Equivalent $Wk_w^2 = W \left( \frac{V}{2\pi N_B} \right)^2$
Kinetic energy of rotating load, ( $KE_r$ ), ft-lb	Inertia of rotating load reflected to brake shaft ( $Wk_b^2$ ), lb-ft <sup>2</sup> Shaft speed at brake ( $N_B$ ), rpm 5875 = Constant	$KE_r = \frac{Wk_b^2 \times N_B^2}{5875}$
Kinetic energy of linear moving load ( $KE_l$ ), ft-lb	Weight of load (W), lb Linear velocity of load (v), ft/sec g = Gravitational acceleration constant, 32.2 ft/sec <sup>2</sup>	$KE_l = \frac{Wv^2}{2g}$
Change in potential energy (PE), ft-lb	Weight of overhauling load (W), lb Distance load travels (s), ft	$PE = Ws$
Total energy absorbed by brake ( $E_T$ ), ft-lb	Total linear kinetic energy, ( $KE_L$ ), ft-lb Total rotary kinetic energy ( $KE_R$ ), ft-lb Potential energy converted to kinetic energy (PE), ft-lb	$E_T = KE_L + KE_R + PE$
Thermal capacity required for rotational or linear moving loads (TC), hp-sec/min	Total system inertia reflected to brake shaft ( $Wk_T^2$ ), lb-ft <sup>2</sup> Shaft speed at brake ( $N_B$ ), rpm Number of stops per minute (n), not less than one $3.2 \times 10^6$ = Constant	$TC = \frac{Wk_T^2 \times N_B^2 \times n}{3.2 \times 10^6}$
Thermal capacity required for overhauling loads (TC), hp-sec/min	Total energy brake absorbs ( $E_T$ ), ft-lb Number of stops per minute (n), not less than one 550 = Constant	$TC = \frac{E_T \times n}{550}$
Linear velocity, ft/min	N = rpm Diameter (D), ft	$V = N\pi D$